

## Useful Work & Exergy Concepts

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ME 407: Advanced Thermodynamics

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## Overview

- 1 Useful Work
- 2 Concept of Exergy
  - Exergy of Heat Transfer
  - Formulation of Useful Work & Exergy
  - Exergy of CM System
  - Exergy of CV System
  - Exergy Concepts: Examples



## Energy: Quantity & Quality

- Energy has both quantity and quality.
  - *Quality of energy* is its potential to produce useful work.
  - **First Law of Thermodynamics:**  
*energy is conserved in all (non-nuclear) processes.*
  - **Second Law of Thermodynamics:**  
*the quality of energy is reduced in all real processes.*
- ⇒ During transformation and transfer, energy is both conserved and degraded.
- Exergy (availability) provides a direct relationship between the thermodynamic state of a system and its capability to do useful work.

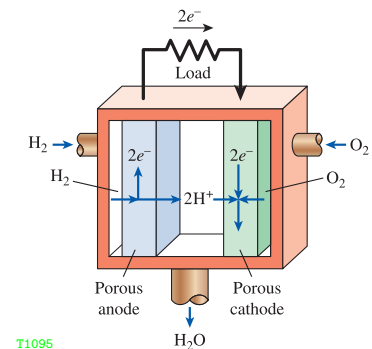


## Datum Condition & Useful Work

**Standard atmosphere:**

$P_0 = 100.325 \text{ kPa}$ ,  $T_0 = 298.15 \text{ K}$

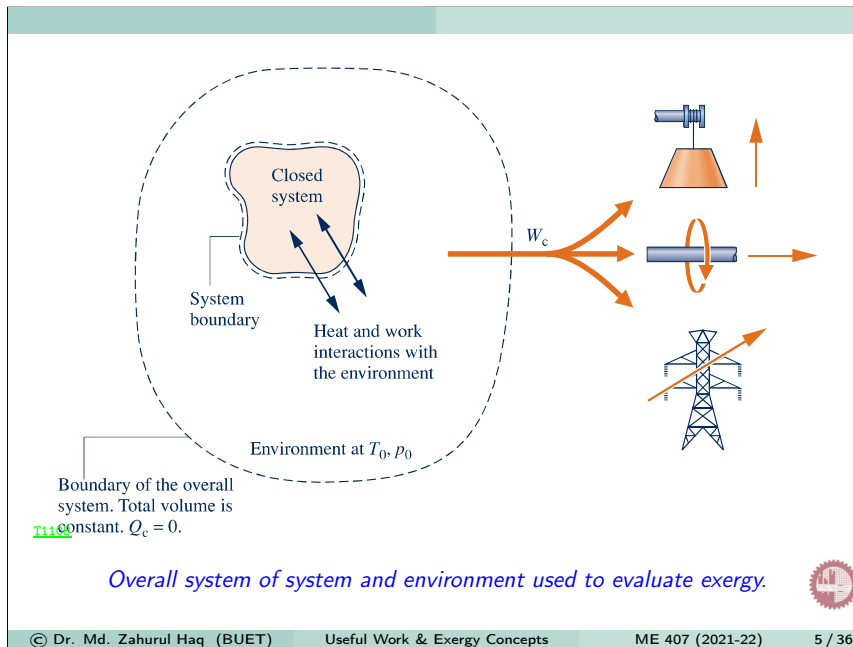
Species	RH = 60%	RH=100%
$N_2$	0.7662	0.7564
$O_2$	0.2055	0.2029
$CO_2$	0.0003	0.0003
$H_2O$	0.0188	0.0313
Other	0.0092	0.0091



T1095

*When the pressure, temperature, composition, velocity, or elevation of a system is different from the environment, there is an opportunity to develop work.*





### Useful Work ( $W_u$ ) & Datum State ( $T_0, P_0$ )

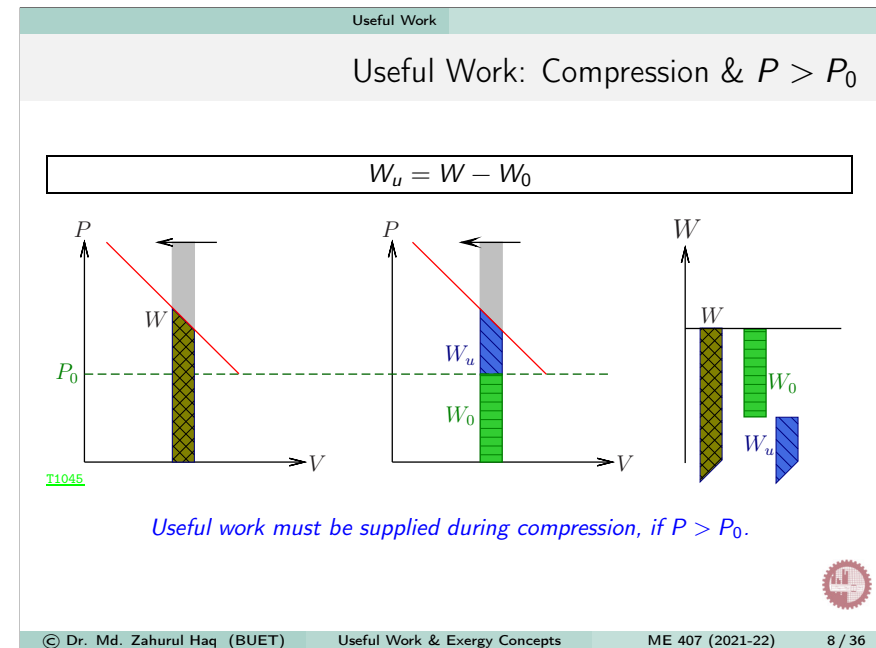
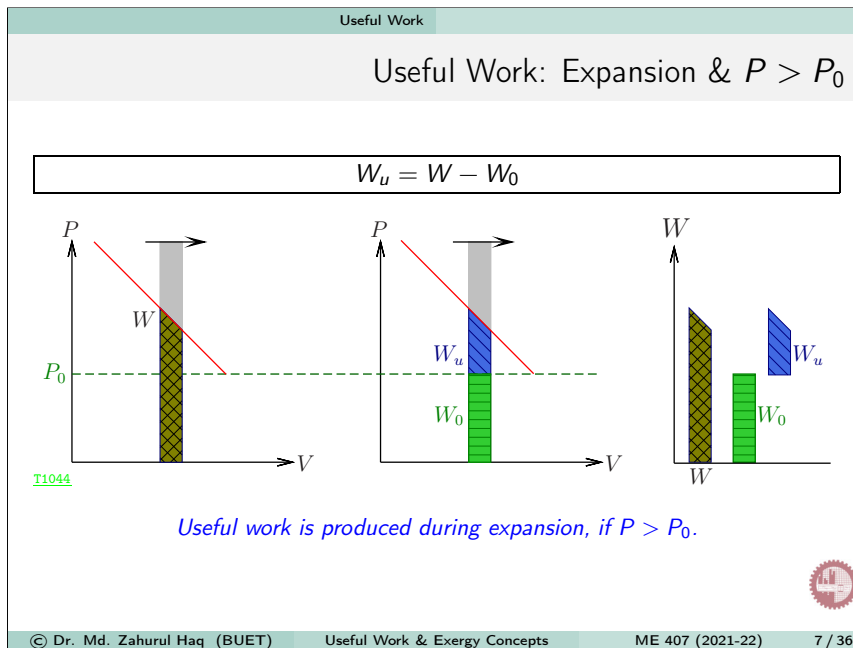
**Datum state:**  $P_0 = 101.325 \text{ kPa}$ ,  $T_0 = 298.15 \text{ K}$

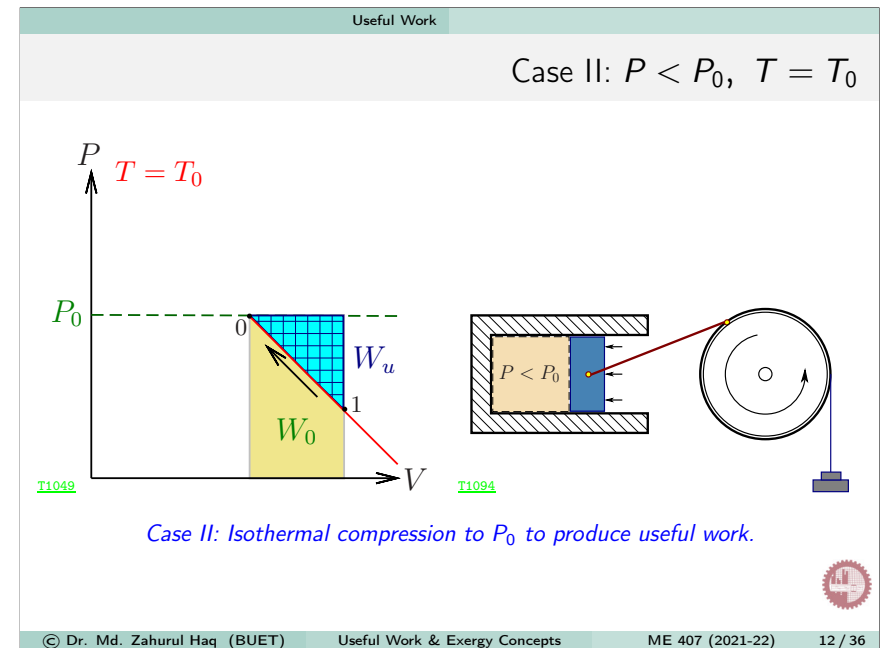
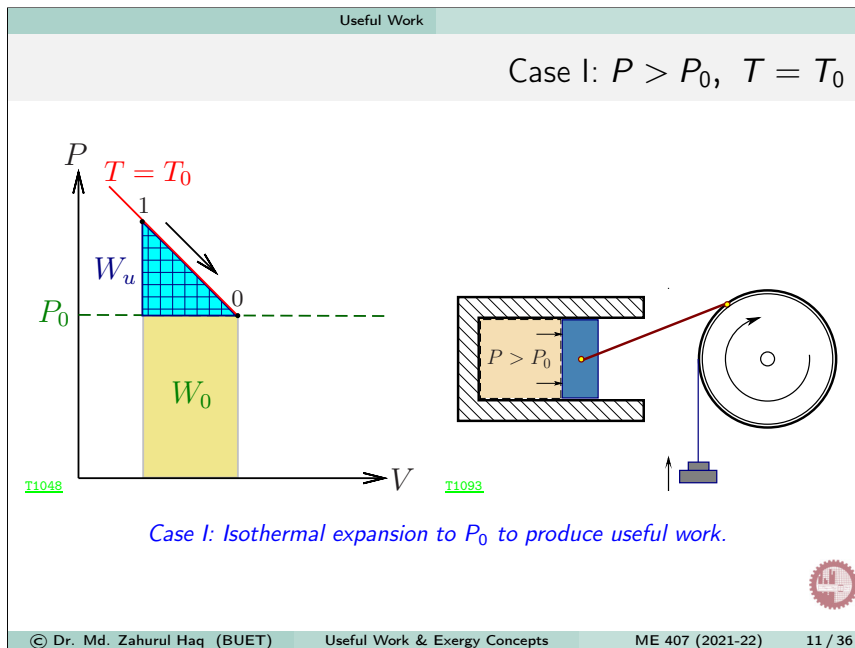
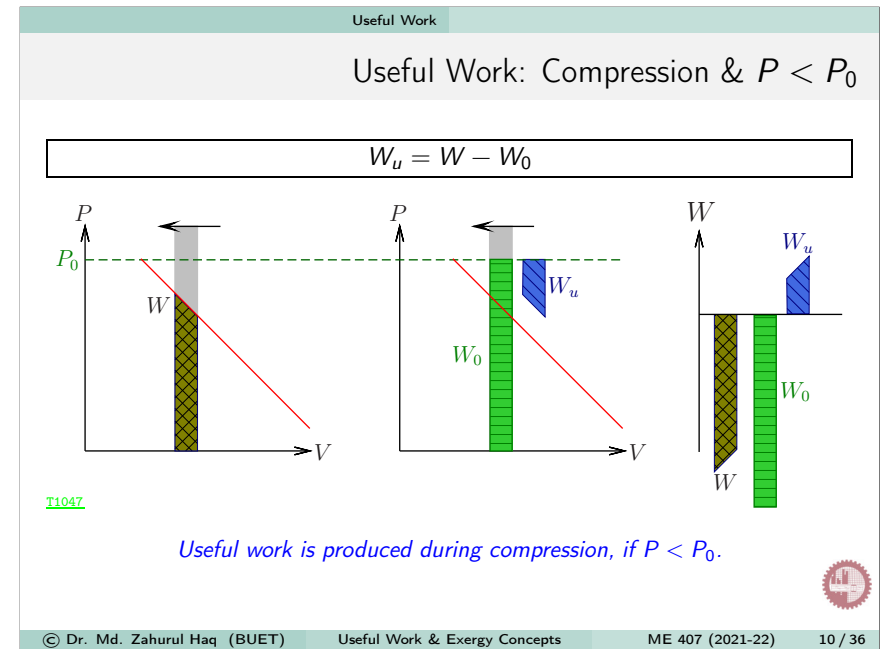
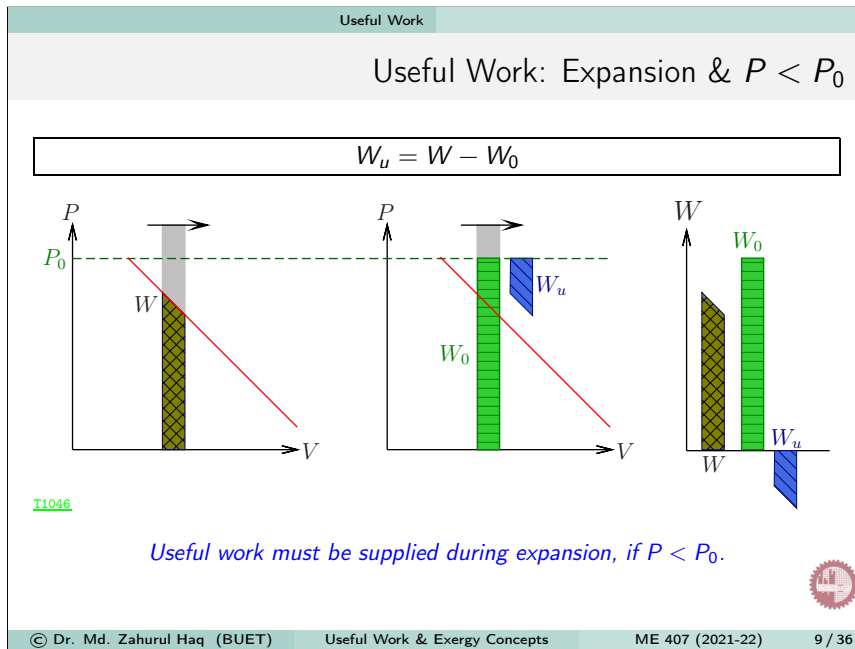
- If  $P \approx P_0 \Rightarrow W = \int_1^2 P dV = P_0 \Delta V \neq 0$ ; But  $W_u = 0$ .
- $\delta W_u = \vec{F} \cdot d\vec{x} = (P - P_0) A dx = (P - P_0) dV = \delta W - P_0 dV$

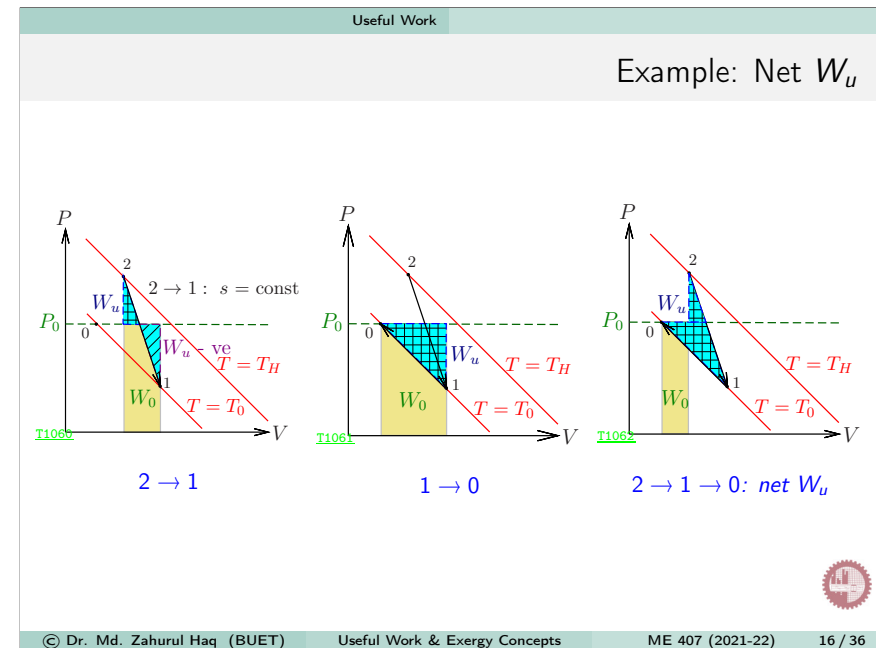
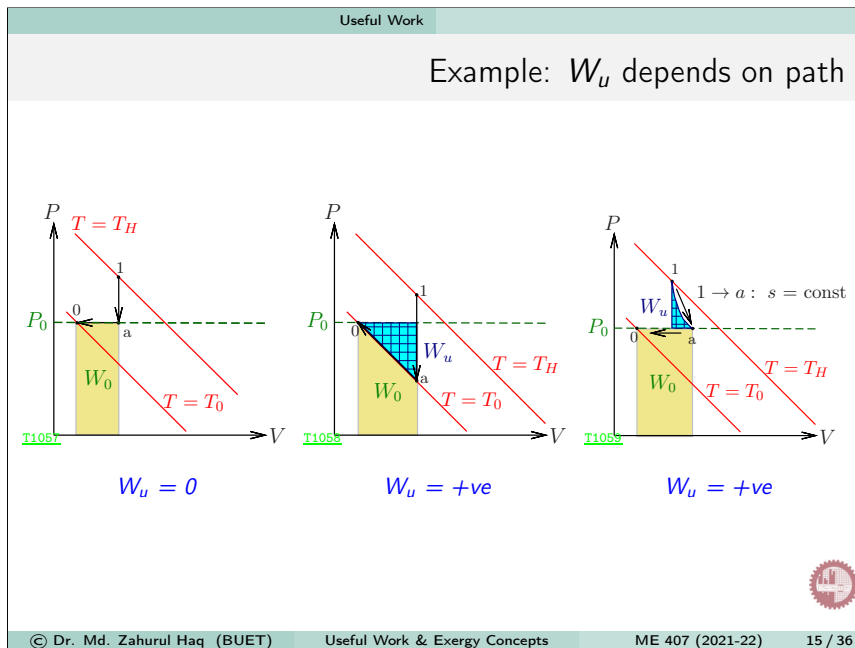
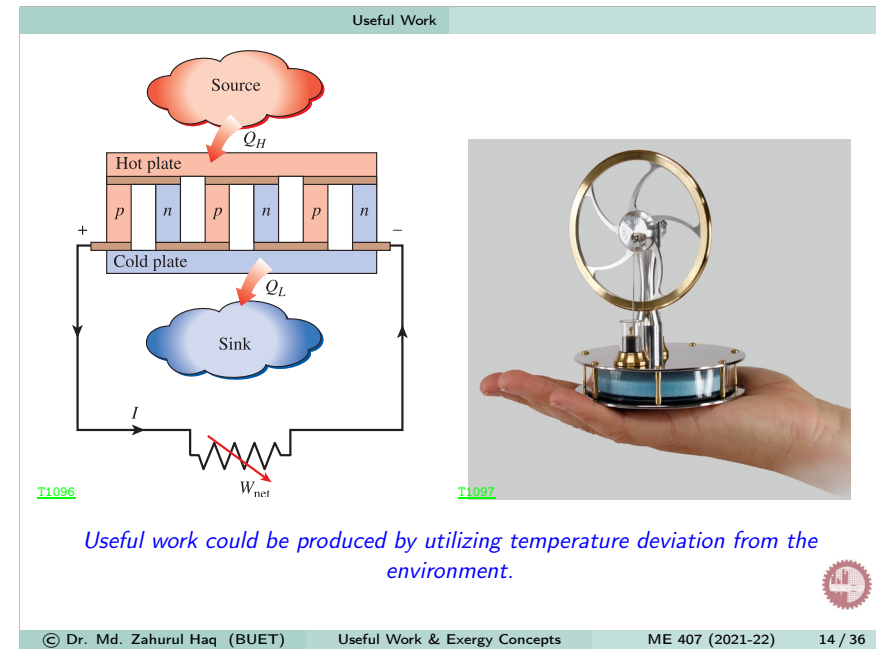
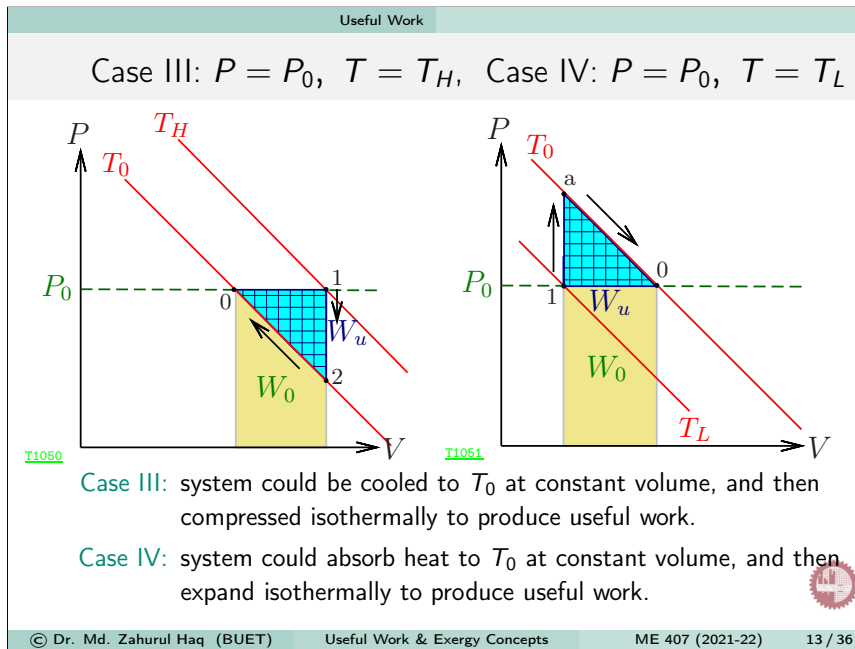
$$\Rightarrow \delta W_u = \delta W - P_0 dV = \delta W - \delta W_0 \Rightarrow W_u = W - W_0$$

- As a closed system expands, some work needs to be done to push the atmospheric air out of the way (and, vice versa)  $\rightsquigarrow W_0$ .

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Concept of Exergy Exergy of Heat Transfer

### Exergy of Heat Transfer ( $\Phi_Q$ )

Reservoir at  $T_R$

Cyclic heat engine

Environment at  $T_0$

$Q$  (Heat input)

$Q_0$  (Heat output)

$W_{rev, H.E.}$  (Work output)

$T$

$T_R$

$T_0$

Area =  $Q$

Area =  $T_0 \Delta S_R$

$\Delta S_R$

$S$

Heat transfer

Entropy transfer

Entropy generated

Exergy transfer

Exergy destroyed

$T_1$

$T_2$

$Q$

$Q/T_1$

$Q/T_2$

$(1 - T_0/T_1)Q$

$(1 - T_0/T_2)Q$

$\Phi_Q \equiv W_{rev, HE} = Q \left(1 - \frac{T_0}{T_R}\right)$

$\Phi_Q = \sum_{j=0}^n Q_j \left(1 - \frac{T_0}{T_j}\right)$

$I_Q = \Phi_{Q,1} - \Phi_{Q,2} = T_0 Q \left[\frac{1}{T_2} - \frac{1}{T_1}\right] = T_0 \sigma_Q$

T1099

Concept of Exergy Exergy of Heat Transfer

$T > T_0$

$Q$  (Heat input)

$\Phi_Q$  (Exergy input)

$T < T_0$

$Q$  (Heat input)

$\Phi_Q$  (Exergy input)

Directions of  $Q$  and  $\Phi_Q$  in relation to  $T$  and  $T_0$

$T$

$\delta Q_j$

$T_j$

$T_0$

$S$

$T$

$P = \text{constant}$

Area =  $\Phi_Q$

Area =  $T_0 \Delta S$

Total area =  $Q$

$S$

$\Phi_Q = \int_1^2 \left(1 - \frac{T_0}{T_j}\right) \delta Q_j$

T1110

Concept of Exergy Exergy of Heat Transfer

### Example: Heat Transfer

Estimate (a) entropy production (b) entropy change (c) loss in work potential, (d) exergy transfers, (e) irreversibility.

Reservoir A

$T_A = 1000 \text{ K}$

Reservoir B

$T_B = 300 \text{ K}$

$\dot{Q} = 10 \text{ kW}$

(a)  $\dot{\sigma}_Q = \frac{\dot{Q}}{T_A T_B} (T_A - T_B) = 0.023 \text{ kW/K}$

(b)  $\frac{dS}{dt} = \frac{dS_A}{dt} + \frac{dS_Q}{dt} + \frac{dS_B}{dt}$

$\Rightarrow \frac{dS_A}{dt} = -\frac{Q_{A, out}}{T_A} = -10/1000 = -0.01 \text{ kW/K}$

$\Rightarrow \frac{dS_B}{dt} = \frac{Q_{B, in}}{T_B} = +10/300 = 0.033 \text{ kW/K}$

$\Rightarrow \frac{dS}{dt} = -0.01 + 0.033 = 0.023 \text{ kW/K}$

$\Rightarrow \frac{dS}{dt} = \dot{\sigma}_Q = 0.023 \text{ kW/K}$

(c)  $\dot{W}_{loss, Q} = T_0 \dot{\sigma}_Q = 300(0.023) = 7.0 \text{ kW}$

(d)  $\Phi_{Q, 1000} = 10 \left(1 - \frac{300}{1000}\right) = 7.0 \text{ kW}$ ,  $\Phi_{Q, 300} = 0$

(e)  $I_Q = \Phi_{Q, 1000} - \Phi_{Q, 300} = 7.0 \text{ kW}$

T1487

Concept of Exergy Formulation of Useful Work & Exergy

### Work & Entropy Production

- ①  $\frac{dE_{cv}}{dt} = \dot{Q}_{cv} - \dot{W}_{cv} + \sum_i \dot{m}_i \left[ h + \frac{V^2}{2} + gz \right]_i - \sum_e \dot{m}_e \left[ h + \frac{V^2}{2} + gz \right]_e$
- ②  $\frac{dS_{cv}}{dt} = \sum \frac{\dot{Q}_j}{T_j} + \sum_i (\dot{m}s)_i - \sum_e (\dot{m}s)_e + \dot{\sigma}_{cv}$ 
  - $\dot{W}_{cv} \equiv$  rate of all possible forms of work transfer.
  - $\delta W_u \equiv \delta W_{cv} - \delta W_0 = \delta W_{cv} - P_0 dV \rightarrow \dot{W}_{cv} = \dot{W}_u + P_0 \frac{dV_{cv}}{dt}$
  - ① - ②  $\times T_0$  : to make the equations dimensionally consistent.

$$\dot{W}_u = \left[ \dot{Q}_{cv} - T_0 \sum_{j=0}^n \frac{\dot{Q}_j}{T_j} \right] - \frac{d}{dt} [E + P_0 V - T_0 S]_{cv} - T_0 \dot{\sigma}_{cv}$$

$$+ \sum_i \dot{m}_i \left[ h + \frac{V^2}{2} + gz - T_0 s \right]_i - \sum_e \dot{m}_e \left[ h + \frac{V^2}{2} + gz - T_0 s \right]_e$$

T1484

Concept of Exergy    Formulation of Useful Work & Exergy

T338

- $\dot{Q}_{cv} = \dot{Q}_0 + \sum \dot{Q}_j = \dot{Q}_0 + \dot{Q}$
- $\sum_{i=0}^n \frac{\dot{Q}_i}{T_j} = \frac{\dot{Q}_0}{T_0} + \sum_{i=1}^n \frac{\dot{Q}_i}{T_j} \approx \frac{\dot{Q}_0}{T_0} + \frac{\dot{Q}}{T_b} : T_b \equiv \text{av. boundary temp.}$

$$\dot{W}_u = \dot{Q} \left[ 1 - \frac{T_0}{T_b} \right] - \frac{d}{dt} [E + P_0 V - T_0 S]_{cv} - T_0 \dot{\sigma}_{cv}$$

$$+ \sum_i \dot{m}_i \left[ h + \frac{V^2}{2} + gz - T_0 s \right]_i - \sum_e \dot{m}_e \left[ h + \frac{V^2}{2} + gz - T_0 s \right]_e$$

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Concept of Exergy    Exergy of CM System

### Exergy of Closed System ( $\dot{m}_i = \dot{m}_e = 0$ )

For CM system:  $\dot{W}_{u,rev} = \dot{Q} \left[ 1 - \frac{T_0}{T_b} \right] - \frac{d}{dt} [E + P_0 V - T_0 S]_{cm}$   
 $\Rightarrow \delta W_{u,rev} = \delta Q \left[ 1 - \frac{T_0}{T_b} \right] - d [E + P_0 V - T_0 S]_{cm}$   
 If system exchanges heat solely with environment at  $T_0$ :

- $\delta W_{u,rev} = -d [E + P_0 V - T_0 S]_{cm} = -d [U + P_0 V - T_0 S]_{cm} = -dA$
- $A \equiv U + P_0 V - T_0 S$      $\Phi \equiv \text{availability of closed system.}$
- For a CM system, from a given state to dead state ( $P_0, T_0$ ):  
 $\Rightarrow W_{u,rev} = -(A_0 - A) = A - A_0$

$\Phi \equiv W_{u,rev,out} = A - A_0$
① $\rightarrow$ ② : $W_{u,rev,1 \rightarrow 2} = -(A_2 - A_1) = -(\Delta U + P_0 \Delta V - T_0 \Delta S) = -\Delta \Phi$
$\phi \equiv \frac{\Phi}{m} \equiv (u - u_0) + P_0(v - v_0) - T_0(s - s_0) = a - a_0$
$w_{u,rev,1 \rightarrow 2} = -\Delta \phi = (\phi_1 - \phi_2)$

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Concept of Exergy    Exergy of CM System

T1100    T1101

Extracting maximum  $W_u$     (a) Energy balance, (b) Entropy balance

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Concept of Exergy    Exergy of CM System

- (Thermo-mechanical/physical) Exergy of a CM system** in a given state is defined as the maximum work output that might be obtained from a system-environment combination as the system proceeds from a given equilibrium state to the **restricted dead state** by a process where any heat-transfer occurs only with the environment.
- At **restricted dead state** the control mass is in thermal and mechanical equilibrium with the environment, but not necessarily in chemical equilibrium with it.
- The difference between the composition of the control mass at restricted dead state and that of the environment can be exploited (by permitting to mix with the environment or enter into chemical reaction with the environmental components) to obtain additional work. The maximum work obtainable in this way is the **chemical exergy**.
- Total exergy is the sum of physical exergy and chemical exergy.

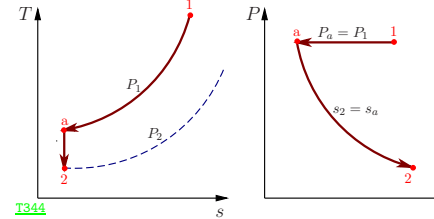
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### Example: Exergy of Air

- $\phi = (u - u_0) + P_0(v - v_0) - T_0(s - s_0)$
- $u - u_0 = c_v(T - T_0)$
- $v = \frac{RT}{P}$
- $s - s_0 = c_v \ln\left(\frac{T}{T_0}\right) + R \ln\left(\frac{v}{v_0}\right) = c_p \ln\left(\frac{T_2}{T_1}\right) - R \ln\left(\frac{P_2}{P_1}\right)$
- Environment:  $T_0 = 298.15 \text{ K}$ ,  $P_0 = 101.325 \text{ kPa}$ .
  - ▷ Air at 298.15 K & 101.325 kPa:  $\phi = 0 \text{ kJ/kg}$
  - ▷ Air at 298.15 K & 50 kPa:  $\phi = 27.4 \text{ kJ/kg}$
  - ▷ Air at 298.15 K & 200 kPa:  $\phi = 16.0 \text{ kJ/kg}$
  - ▷ Air at 200 K & 101.325 kPa:  $\phi = 20.8 \text{ kJ/kg}$
  - ▷ Air at 400 K & 101.325 kPa:  $\phi = 14.4 \text{ kJ/kg}$

When the pressure, temperature, composition, velocity, or elevation of a system is different from the environment, there is an opportunity to develop work.

### Winterbone Ex. 2.3



- $P_1 = 200 \text{ kPa}$ ,  $T_1 = 550 \text{ K}$
- $P_2 = P_0 = 100 \text{ kPa}$ ,  $T_2 = T_0 = 300 \text{ K}$
- ⇒  $T_a = T_2 \left[\frac{P_a}{P_2}\right]^{(k-1)/k} = 366 \text{ K}$

- $v = RT/P \Rightarrow v_1 = 0.7896$ ,  $v_a = 0.5248$  &  $v_2 = 0.861 \text{ m}^3/\text{kg}$
- $w_{sys}|_{1 \rightarrow a} = \int_1^a P dv = P_1(v_a - v_1) = -52.96 \text{ kJ/kg}$
- $w_{surr}|_{1 \rightarrow a} = \int_1^a P_0 dv = P_0(v_a - v_1) = -26.46 \text{ kJ/kg}$
- $w_u|_{1 \rightarrow a} = w_{sys}|_{1 \rightarrow a} - w_{surr}|_{1 \rightarrow a} = -26.46 \text{ kJ/kg}$
- $w_{sys}|_{a \rightarrow 2} = \int_a^2 P dv = \frac{R}{1-k}(T_2 - T_a) = +47.14 \text{ kJ/kg}$
- $w_{surr}|_{a \rightarrow 2} = \int_a^2 P_0 dv = P_0(v_2 - v_a) = +33.63 \text{ kJ/kg}$
- $w_u|_{a \rightarrow 2} = w_{sys}|_{a \rightarrow 2} - w_{surr}|_{a \rightarrow 2} = +13.51 \text{ kJ/kg}$

- $w_u|_{1 \rightarrow 2} = w_u|_{1 \rightarrow a} + w_u|_{a \rightarrow 2} = -12.95 \text{ kJ/kg}$
- $1 \rightarrow a$ : temperature is decreased from 550 K to 365.7 K and heat transfer to surroundings occurs. It can be used to produce useful work by reversible heat engine.
  - $\delta w_R = -\eta_R \delta q = -(1 - T_0/T) \delta q$
  - $w_R|_{1 \rightarrow a} = -\int_1^a (1 - T_0/T) \delta q = -C_p \int_1^a (1 - T_0/T) dT = +62.18 \text{ kJ/kg}$
  - $w_R|_{a \rightarrow 2} = 0 \text{ kJ/kg}$  as no heat transfer occurs.
  - $w_R|_{1 \rightarrow 2} = w_R|_{1 \rightarrow a} + w_R|_{a \rightarrow 2} = +62.18 \text{ kJ/kg}$
- $w_{u,max}|_{1 \rightarrow 2} = w_u|_{1 \rightarrow 2} + w_R|_{1 \rightarrow 2} = +49.23 \text{ kJ/kg}$
- **Exergy approach:**

$$w_{u,max}|_{1 \rightarrow 2} = -\Delta\phi = -[(u_2 - u_1) + P_0(v_2 - v_1) - T_0(s_2 - s_1)]$$
  - $u_2 - u_1 = c_v(T_2 - T_1) = -179.5 \text{ kJ/kg}$
  - $P_0(v_2 - v_1) = 7.14 \text{ kJ/kg}$
  - $T_0(s_2 - s_1) = T_0 \left\{ c_p \ln\left(\frac{T_2}{T_1}\right) - R \ln\left(\frac{P_2}{P_1}\right) \right\} = -123.7 \text{ kJ/kg}$
  - $w_{u,max}|_{1 \rightarrow 2} = +49.28 \text{ kJ/kg}$

### Exergy Balance of a CM System

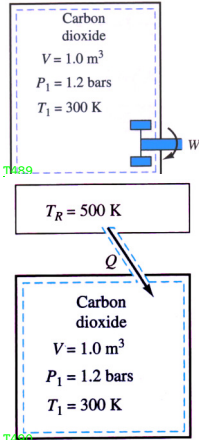
- $\delta W_u = \delta Q \left[1 - \frac{T_0}{T_b}\right] - d[E + P_0V - T_0S]_{cv} - T_0\delta\sigma_{cm}$
- $$W_u = \int_1^2 \left[1 - \frac{T_0}{T_b}\right] \delta Q - [\Delta U + P_0\Delta V - T_0\Delta S] - T_0\sigma_{cm}$$
- $$= \Phi_Q - \Delta\Phi - I_{cm}$$

$$\Delta\Phi = \Phi_Q - W_u - I_{cm}$$

- $\Delta\Phi$  = exergy change of a CM system
- $\Phi_Q$  = exergy transfer with heat transfer into system
- $W_u$  = exergy transfer with useful work out of system
- $I_{cm} \equiv T_0\sigma_{cm}$  = exergy destruction within CM

- $W_{u,rev} = W_u|_{I_{cm}=0} = \Phi_Q - \Delta\Phi \Rightarrow I_{cm} = W_{u,rev} - W_u$
- Heat transfer region:  $\Rightarrow I_Q = \Phi_{Q,in} - \Phi_{Q,out} \because \Delta\Phi = W_u = 0$
- For isolated system:  $\Rightarrow \Delta\Phi = -I_{cm} \rightarrow \Delta\Phi \leq 0$

**Wark(1999), Ex. 9.5:** ▷ Temperature is increased to 400 K using (a) using a paddle wheel (b) using heat transfer from a source at 500 K. Irreversibility = ?



- $R = R_u/M = 188.9 \text{ J/kg}\cdot\text{K}$ ,  $m = \frac{P_1 V_1}{R T_1} = 2.12 \text{ kg}$
- $\Phi \equiv (u - u_0) + P_0(v - v_0) - T_0(s - s_0)$
- $\Delta\Phi = \Phi_Q - W_u - I_{cm}$
- **(a):**  $W_u = W_{12} = m(u_2 - u_1) = -149.1 \text{ kJ}$
- $\Phi_Q = 0.0$ : no heat transfer
- $\Delta\Phi = 20.4 \text{ kJ/kg} \Rightarrow I_{cm} = 128.7 \text{ kJ} \blacktriangleleft$
- **(b):**  $T_b = T_R = 500 \text{ K}$ ,  $Q_{12} = m(u_2 - u_1) = 149.1 \text{ kJ}$
- $W_u = W_{12} = 0.0$
- $\Phi_Q = Q_{12}(1.0 - T_0/T_b) = 59.6 \text{ kJ}$
- $\Delta\Phi = 20.4 \text{ kJ/kg} \Rightarrow I_{cm} = 39.2 \text{ kJ} \blacktriangleleft$

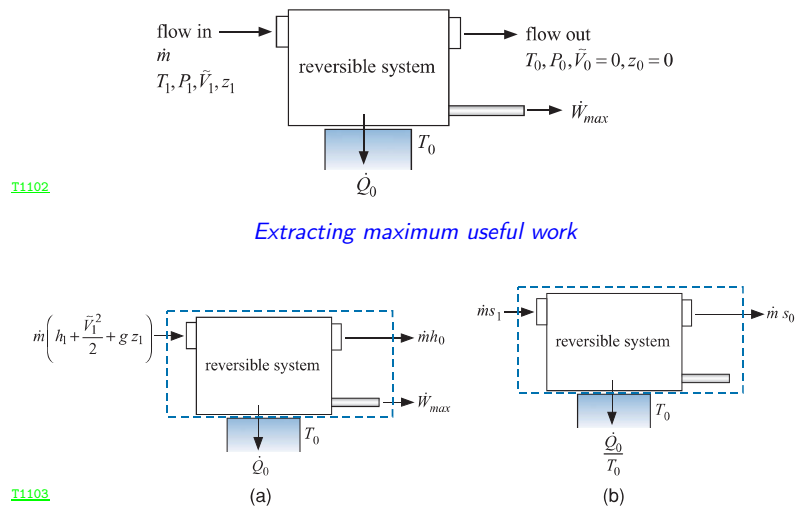
### Exergy of CV System

$$\dot{W}_u = \sum_{j=1}^n \dot{Q}_j \left[ 1 - \frac{T_0}{T_j} \right] - \frac{d}{dt} [E + P_0 V - T_0 S]_{cv} - T_0 \dot{\sigma}_{cv} + \sum_i \dot{m}_i \left[ h + \frac{V^2}{2} + gz - T_0 s \right]_i - \sum_e \dot{m}_e \left[ h + \frac{V^2}{2} + gz - T_0 s \right]_e$$

- For SSSF process:  $\frac{d}{dt} [E + P_0 V - T_0 S]_{cv} = 0$  &  $P_0 \Delta V = 0$ .
- ⇒  $\dot{W}_u = \dot{W}_{sf}$  &  $\dot{W}_{sf,rev} = \dot{W}_{sf} + T_0 \dot{\sigma}_{cv}$
- **Flow exergy ( $\psi$ )** of a fluid in steady flow is defined as the maximum work output that can be obtained as the fluid is changed reversibly from the given state to a dead state in a process where any heat transfer occurs solely with the environment.

$$\psi \equiv w_{u,rev} \equiv (h - h_0) - T_0(s - s_0) + \frac{V^2}{2} + gz$$

$$\dot{W}_{sf,rev} = \sum_{j=1}^n \dot{Q}_j \left[ 1 - \frac{T_0}{T_j} \right] + \sum_i \dot{m}_i \psi_i - \sum_e \dot{m}_e \psi_e$$



(a) Energy balance, (b) Entropy balance

### Exergy Balance of a CV System: SSSF Process

$$\dot{W}_{u,act} = \sum_{j=1}^n \dot{Q}_j \left[ 1 - \frac{T_0}{T_j} \right] - \frac{d}{dt} [E + P_0 V - T_0 S]_{cv} - T_0 \dot{\sigma}_{cv} + \sum_i \dot{m}_i \left[ h + \frac{V^2}{2} + gz - T_0 s \right]_i - \sum_e \dot{m}_e \left[ h + \frac{V^2}{2} + gz - T_0 s \right]_e$$

$$\Rightarrow \dot{W}_{u,act} = \Phi_Q - 0 - \dot{I}_{cv} + \sum_i \dot{m}_i \psi_i - \sum_e \dot{m}_e \psi_e$$

$$\Rightarrow \Delta(m\psi) = \sum_e \dot{m}_e \psi_e - \sum_i \dot{m}_i \psi_i = \dot{\Phi}_Q - \dot{W}_{u,act} - \dot{I}_{cv}$$

- $\Delta(m\psi)$  : Net exergy outflux rate with mass flow.
- $\dot{\Phi}_Q$  : Net exergy transfer rate with heat transfer into CV.
- $\dot{W}_{u,act}$  : Net exergy transfer rate with work out of CV.
- $\dot{I}_{cv}$  : Net exergy destruction rate within CV.



Concept of Exergy      Exergy of CV System

Energy In	Energy Out
90 MW (heat transfer)	40 MW (power)
10 MW (at inlet <i>i</i> )	60 MW (at exit <i>e</i> )
<u>100 MW</u>	<u>100 MW</u>

(a)

Exergy In	Exergy Out
60 MW (heat transfer)	40 MW (power)
2 MW (at inlet <i>i</i> )	15 MW (at exit <i>e</i> )
<u>62 MW</u>	<u>55 MW</u>

Exergy Destroyed = 62 MW - 55 MW = 7 MW  
(b)

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Comparing energy and exergy for a control volume at steady state. (a) Energy analysis. (b) Exergy analysis.

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Concept of Exergy      Exergy Concepts: Examples

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Concept of Exergy      Exergy Concepts: Examples

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Concept of Exergy      Exergy Concepts: Examples

## Characteristics of Exergy

- Exergy is the maximum work that can be extracted from a combined system and the environment.
- At datum state, both system and the environment possess energy, but the exergy is zero.
- Exergy is never negative.
- Exergy is a measure of deviation from the state of the system from that of the environment. The greater the deviation, the higher the value of exergy.
- Exergy is destroyed by irreversibilities.

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