

# Plate & Frame Heat Exchanger (PFHX)

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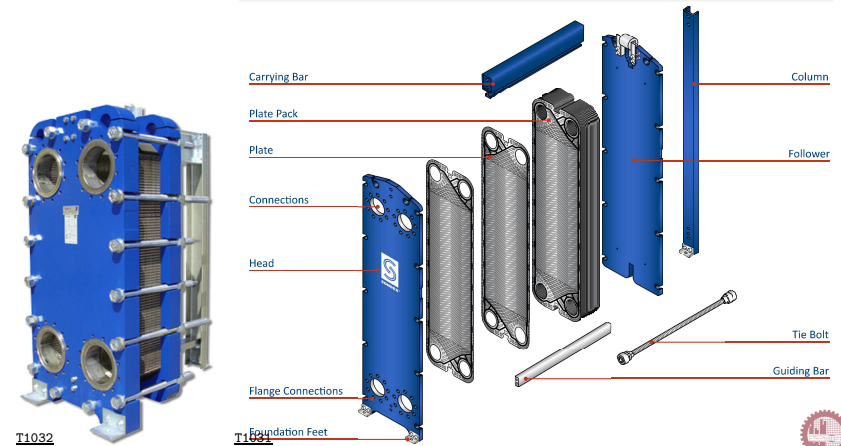
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ME 307: Heat Transfer Equipment Design  
<http://zahurul.buet.ac.bd/ME307/>

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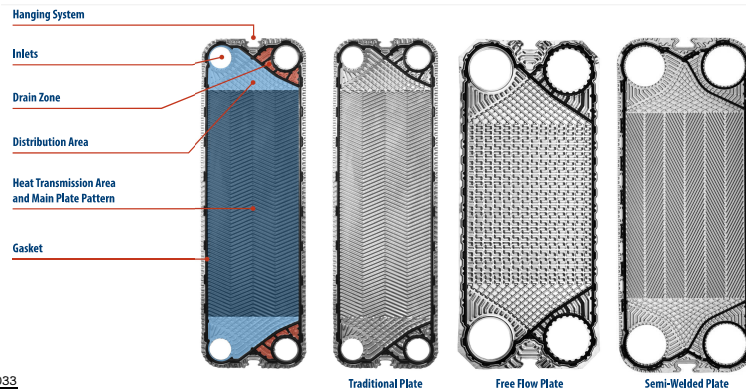
### Plate & Frame Heat Exchanger (PFHX)



T1032

T1034

## Plate & Frame Heat Exchanger (PFHX)



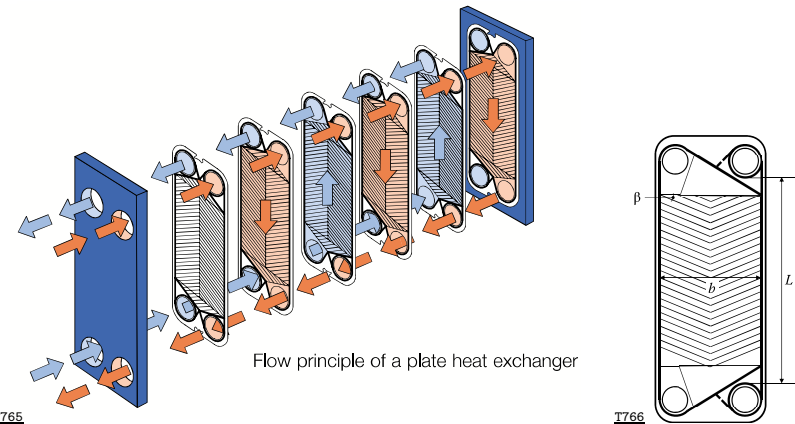
T1033

Traditional Plate

Free Flow Plate

Semi-Welded Plate

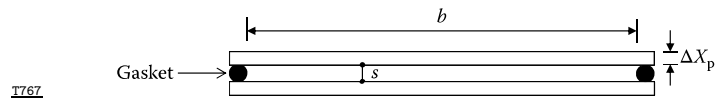
## Plate & Frame Heat Exchanger (PFHX)



T765

T766

- $NTU = \frac{U_o A_o N_s}{(\dot{m} C_p)_{min}}$ ,  $A_o = bl$
- Effective  $NTU = F_{corr} NTU$ ,  $F_{corr} = 1 - 0.0166 NTU$
- Heat transfer & pressure loss correlations are based on the hydraulic diameter,  $D_h$  of the passage formed by the two plates.



- $\frac{1}{U_o} = \frac{1}{h_h} + \frac{1}{h_c} + \frac{\Delta X_p}{k} + R_{di} + R_{do}$
- Flow velocity,  $V = \begin{cases} \frac{2\dot{m}}{\rho s b (N_s + 1)} & : \text{odd number of plates} \\ \frac{2\dot{m}}{\rho s b N_s}, \frac{2\dot{m}}{\rho s b (N_s + 2)} & : \text{even number of plates} \end{cases}$
- $D_h = \frac{4(\text{cross-sectional area})}{\text{wetted perimeter}} = \frac{4sb}{2s+2b} \approx \frac{4sb}{2b} = 2s, \quad Re = \frac{VD_h}{\nu}$
- Laminar flow:  $Nu = 1.86(Gz)^{1/3} \left(\frac{\mu_b}{\mu_w}\right)^{0.14}; \quad Gz = \frac{RePr}{L/D} \quad Re < 100$
- Turbulent flow:  $Nu = 0.374Re^{0.668}Pr^{1/3}; \quad Re > 100$
- Friction factor,  $f = \begin{cases} \frac{280}{Re} & : 1 < Re < 10 \\ \frac{100}{Re^{0.589}} & : 10 < Re < 100 \\ \frac{12}{Re^{0.183}} & : Re > 100 \end{cases}$
- Pressure drop,  $\Delta P = \frac{fL}{D_h} \left(\frac{1}{2}\rho V^2\right) + 1.3\left(\frac{1}{2}\rho V_p^2\right) \approx \frac{fL}{D_h} \left(\frac{1}{2}\rho V^2\right)$



- Flexibility of design through a variety of plate sizes and pass arrangements
- Easily accessible heat transfer area, permitting changes in configuration to suit changes in processes requirements through changes in the number of plates
- Efficient heat transfer; high heat transfer coefficients for both fluids because of turbulence and a small hydraulic diameter
- Very compact, and low in weight; in spite of their compactness, 1500 m<sup>2</sup> of surface area is available in a single unit
- Only the plate edges are exposed to the atmosphere; the heat losses are negligible and no insulation is required



**PFHX Rating** ▷ 0.975 kg/s water at 25°C is to be cooled using 1.0 kg/s water available at 7°C. If  $b = 457$  mm,  $L = 914$  mm,  $s = 5.08$  mm,  $t = 1.016$  mm,  $k = 14.3$  W/mK,  $N_s = 11$ , fouling coefficients for both fluids =  $3.52 \times 10^{-6}$ , rate the heat-exchanger.

